

CH330 Fall 2021 Final Exam KEY

- 1) a) $U_\infty = 0$ heat transfer can only occur by conduction. Consider a sphere with Radius R transferring heat to surroundings when $U_\infty = 0$ we can write (for $q|_{r=R}$)



$$-k \left. \frac{\partial T}{\partial r} \right|_{r=R} = h (T|_{r=R} - T_\infty)$$

$T_\infty @ r = \infty$

Recall that $Nu_D = \frac{hD}{k}$

we can write a steady state differential balance for heat conduction in "surroundings" and solve for "h" as follows:

$0 = k \nabla^2 T$ or since $T = T(r)$ in spherical coordinates

$0 = \frac{k}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right)$ with B.C.'s as: $\begin{cases} T(r=R) = T_s \\ T(r=\infty) = T_\infty \end{cases}$

$\Downarrow$
 $r^2 \frac{dT}{dr} = C_1$

$\frac{dT}{dr} = \frac{C_1}{r^2}$

$T = C_2 - \frac{C_1}{r}$

apply B.C.'s

$T_\infty = C_2$

$T_s = T_\infty - \frac{C_1}{R}$

$\Downarrow$

$R(T_s - T_\infty) = C_1$

thus

$$T = T_\infty + \frac{R(T_s - T_\infty)}{r}$$

then we can solve for $g|_{r=R}$

$$\begin{aligned} g|_{r=R} &= -k \left. \frac{dT}{dr} \right|_{r=R} \\ &= \frac{+kR(T_s - T_\infty)}{R^2} \\ &= \frac{k(T_s - T_\infty)}{R} \end{aligned}$$

we can equate this to $h(T_s - T_\infty)$

$$\frac{k(T_s - T_\infty)}{R} = h(T_s - T_\infty)$$

thus $h = \frac{k}{R}$

Plugging this into $Nu = \frac{hD}{k}$

$$Nu = \frac{\frac{k}{R} \cdot D}{k} = \frac{\cancel{k} \cdot 2R}{\cancel{k}} = 2 \quad \checkmark$$

2) in the glass continuum $T = T(z)$
Steady state, no heat production.

thus $0 = k \nabla^2 T$ or $0 = \nabla^2 T$ in 1D then

$0 = \frac{d^2 T}{dz^2}$ with Boundary conditions! }

① $z=0$

$$q|_{z=0} = h_o (T|_{z=0} - T_o)$$

OR

$$-k_g \frac{dT}{dz} = h_o (T|_{z=0} - T_o) \quad \}$$

② $z=L$ the heat produced will be both
conducted into glass and convected
into air thus we can write

$$S_A = |q_{\text{cond}}| + |q_{\text{conv}}|$$

OR

$$S_A = k_g \frac{dT}{dz} + h_i (T|_{z=L} - T_i) \quad \times$$

Solve differential eq. $\frac{dT}{dz} = C_1$

$$T = C_1 z + C_2 \quad \leftarrow$$

apply B.C. @ $z=0$

$$-k_g C_1 = h_o (C_2 - T_o)$$

apply B.C. @ $z=L$

$$S_A = k_g C_1 + h_i [C_1 L + C_2 - T_i]$$

from the eq. (a) $z=0$

$$-k_g C_1 = h_o C_2 - h_o T_o \Rightarrow C_2 = \frac{h_o T_o - k_g C_1}{h_o}$$

or

$$C_2 = T_o - \frac{k_g}{h_o} C_1$$

then plug into eq. for $z=L$

$$S_A = k_g C_1 + h_i \left[C_1 L + \frac{h_o T_o - k_g C_1}{h_o} - T_c \right]$$

expand

$$S_A = k_g C_1 + h_i L C_1 + h_i T_o - \frac{k_g h_i}{h_o} C_1 - h_i T_c$$

Rearrange!

$$S_A + h_i (T_c - T_o) = C_1 \left[k_g + h_i L - \frac{h_i k_g}{h_o} \right]$$

thus

$$C_1 = \frac{S_A + h_i (T_c - T_o)}{k_g + h_i L - \frac{h_i k_g}{h_o}}$$

and

$$C_2 = T_o - \frac{k_g}{h_o} \left[\frac{S_A + h_i (T_c - T_o)}{k_g + h_i L - \frac{h_i k_g}{h_o}} \right]$$

Finally!

$$T(z) = C_1 z + C_2$$

with C_1 & C_2 as written above!

3)

$$k = 6 \text{ W m}^{-1} \text{ K}^{-1}$$

$$c_p = 850 \text{ J kg}^{-1} \text{ K}^{-1}$$

$$\rho = 5200 \text{ kg m}^{-3}$$

$$h = 500 \text{ W m}^{-1} \text{ K}^{-1}$$

$$T_{\infty} = 95^{\circ}\text{C}$$

$$T_i = 18^{\circ}\text{C}$$

$$T(r=?) = 44^{\circ}\text{C}$$

$$7 \text{ mm}$$

$$\text{1st calculate } B_{iL} = \frac{hL}{k} = \frac{500 \cdot 0.007}{6} = 0.5833$$

$$\text{then from tables: } \lambda_1 = 0.69 \quad A_1 = 1.08$$

$$\text{at the "center"} \quad \theta = \frac{T_0 - T_{\infty}}{T_i - T_{\infty}} = A_1 e^{-\lambda_1^2 \tau} \quad \tau > 0.2$$

$$\frac{44 - 95}{18 - 95} = \frac{-51}{-77} = 0.662 = 1.08 \exp(-0.69^2 \tau)$$

$$+ 0.4889 = +0.69^2 \tau$$

$$\tau = 1.03 \quad (> 0.2!) \quad \checkmark$$

$$\tau = \frac{\alpha t}{L^2}$$

$$\alpha = \frac{k}{\rho c_p} = \frac{6}{5200 \cdot 850} = 1.357 \times 10^{-6}$$

$$t = \frac{\tau L^2}{\alpha} = \frac{1.03 \cdot (0.007)^2}{1.357 \times 10^{-6}} = 37.2 \text{ sec}$$

4 or 4) mass exchanger

in analogy to heat transfer.

$$N = \sqrt{\frac{4 k_m L^2}{D \mathcal{D}}}$$

where k_m is some mass transfer coefficient
and $\mathcal{D}$ is some diffusion coefficient.

the units of $\frac{m^2}{s}$ and N is adimensional

then

$$\frac{\left[\frac{m}{s}\right] \left[m^2\right]}{\left[m\right] \left[\frac{m^2}{s}\right]}$$

k_m must have units $\frac{m}{s}$

then

$$N = \sqrt{\frac{4 k_{c,L} L^2}{D \mathcal{D}_{CO_2, \text{calc. carb.}}}}$$

where

$k_{c,L}$ is the mass transfer coefficient

Based on the liquid phase with respect
to concentration

and

$\mathcal{D}_{CO_2, \text{calc. carb.}}$ is the diffusion coefficient
for CO_2 in the calcium carbonate

5) Drying Methane

① $p = 1.2 \text{ atm}$
methane
 $T_{db} = 30^\circ\text{C}$
dew point $= 21.2^\circ\text{C}$



②
1.09 kg/s
0.008 kg H₂O
kg dry CH₄

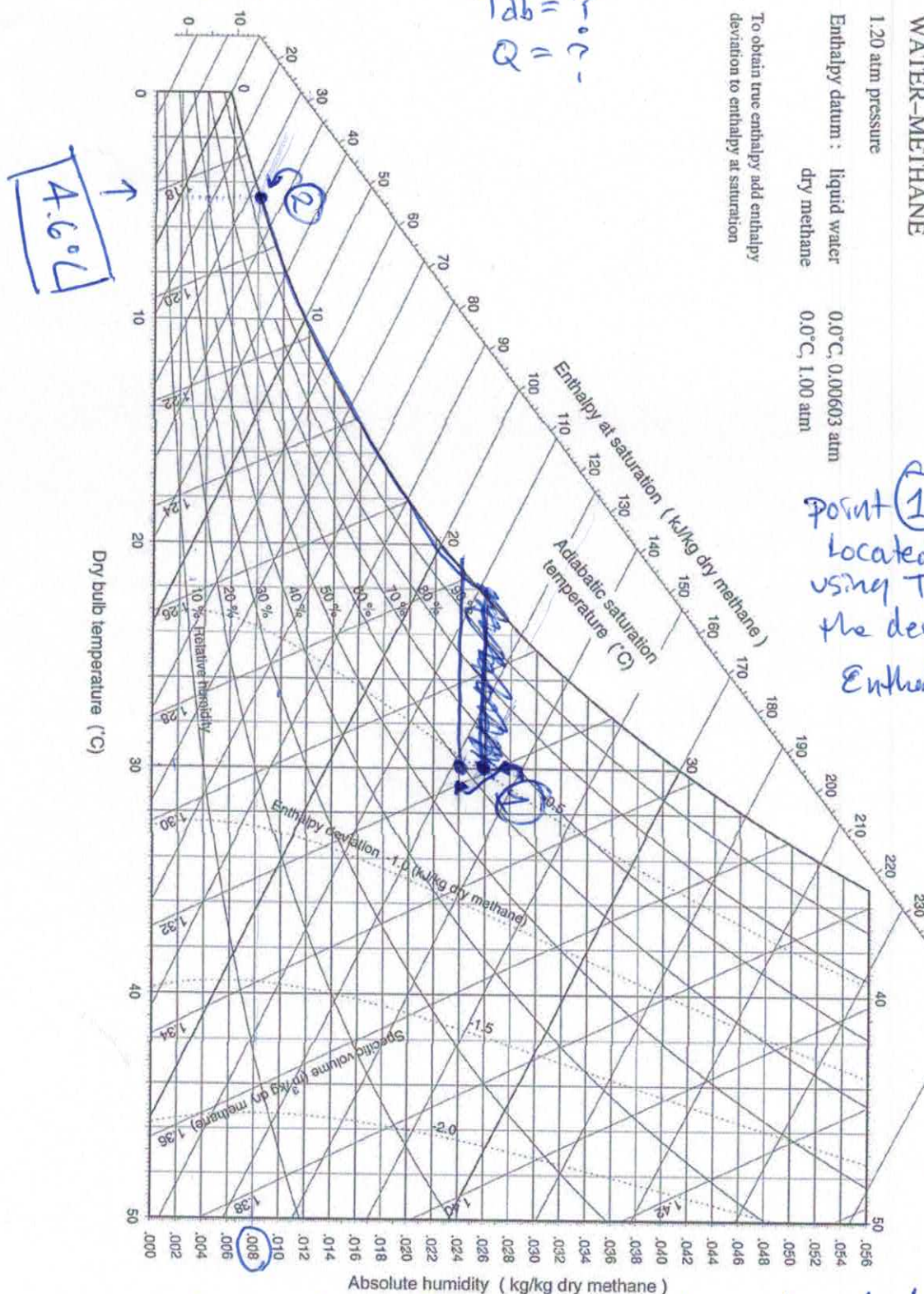
$T_{db} = ?$
 $Q = ?$

To obtain true enthalpy add enthalpy deviation to enthalpy at saturation

WATER-METHANE	
1.20 atm pressure	
Enthalpy datum: liquid water	0.0°C, 0.00603 atm
dry methane	0.0°C, 1.00 atm

point ① can be located on the chart using T_{db} and considering the dew point.

Enthalpy $3125 \frac{\text{kJ}}{\text{kg dry methane}}$



upon exiting the cooler the CH₄ will be saturated thus point ② must be as indicated, where $T_{db} = 4.6^\circ\text{C}$ enthalpy $= 30 \frac{\text{kJ}}{\text{kg}}$

$$\left[125 \frac{\text{kJ}}{\text{kg}} - 30 \frac{\text{kJ}}{\text{kg}} \right] 1.09 \frac{\text{kg}}{\text{s}} = 103.6 \frac{\text{kJ}}{\text{s}} = 104 \text{ kW}$$